

An easy calculation for Heat-sink size is explained on this subject.
Please perform actual evaluation and confirmation to determine
final heat-sink construction.

1. Calculation of Power Loss from o/p power & Efficiency

Example : Output — $V_o=48V, I_o=10A (P_o=480W)$ Input — $V_{in}=200VAC$

Efficiency is $\eta = 85\%$ * according to "PFE500-Series Evaluation Data" at the condition above.

So, power loss at PF500 unit is calculated : $P_d = P_o \times (\frac{1}{\eta} - 1) = 480W \times (1/0.85 - 1) = 85W$

This means that Base-plate Temperature increases $\Delta T_p = 85^\circ C$ when attaching
a heat-sink with $1^\circ C/W$ thermal resistance.

$$(\Delta T_p = \theta_{hs-a} \times P_d = 1^\circ C/W \times 85W)$$

* with 1% margin

2. Definition of Base-plate Temperature Increase (ΔT_p) from Ambient Temperature (T_a)

In this case, when Ambient $T_a = 50^\circ C$ max., $T_p = 90^\circ C$ max. for customer equipment,

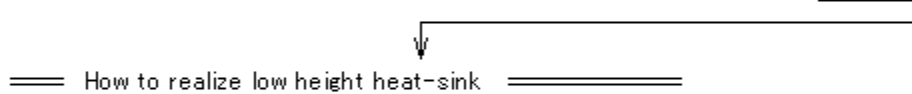
then $\Delta T_p = 90 - 50 = 40^\circ C$ max., is allowed.

3. Calculation of Thermal Resistance (θ_{hs-a}) for Heat-sink

$$\Delta T_p = 40^\circ C, P_d = 85W \longrightarrow \theta_{hs-a} = \frac{\Delta T_p}{P_d} = \underline{0.47^\circ C/W}$$

4. Study of Actual Heat-sink

First, when considering standard heat-sink of DENSEI-LAMBDA "HAF-15T", more than 2 m/s airflow
velocity is required by the graph below. But 15T height is too high for recent "low profile" requirements.



Generally, heat-sink with same enveloping volume has same thermal resistance.

So, for example, by changing its size as 2 times of "D" and a half of "H" below,
same thermal resistance can be achieved.

